

What is claimed is:

1. A control apparatus for four wheel drive vehicle having a differential limiting unit on at least either
5 one of a front wheel side or a rear wheel side, comprising:
 - a turning state determining unit for determining a turning state of said vehicle;
 - an actual left and right wheel differential speed calculating unit for calculating an actual left and right
10 wheel differential speed in accordance with a turning state of the vehicle;
 - a target differential speed setting unit for setting a left and right wheel target differential speed;
 - a differential limiting torque calculating unit for
15 setting a differential limiting torque at 0 on basis of information from the differential limiting unit when an inside wheel speed falls below an outside wheel speed by a preset threshold value in a turning state and for calculating a differential limiting torque on basis of
20 said information about the left and right wheel target differential speed and the actual left and right wheel differential speed when said inside wheel speed exceeds said outside wheel speed by a preset threshold value in said turning state; and
25 a front and rear wheel differential limiting torque

setting unit for setting a front and rear wheel differential limiting torque calculated in the differential limiting torque calculating unit in dependency on said differential limiting torque on basis
5 of said information.

2. The control apparatus as set forth in Claim 1, wherein:

0 is the threshold value used for comparing the inside
10 wheel speed with the outside wheel speed in the turning state.

3. The control apparatus as set forth in Claim 1, wherein:

15 the actual left and right wheel differential speed calculating unit regards an actual left and right wheel differential speed as an absolute value of said left and right wheel differential speed after the differential limiting torque calculating unit calculates a
20 differential limiting torque based on said information from the differential limiting unit in dependency on the left and right wheel target differential speed and the absolute value when the turning state determining unit determines said turning state substantially as said
25 running straight condition.

4. The control apparatus as set forth in Claim 2,
wherein:

the actual left and right wheel differential speed
5 calculating unit regards an actual left and right wheel
differential speed as an absolute value of said left and
right wheel differential speed after the differential
limiting torque calculating unit calculates a
differential limiting torque based on said information
10 from the differential limiting unit in dependency on the
left and right wheel target differential speed and the
absolute value when the turning state determining unit
determines said turning state substantially as said
running straight condition.

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5. The control apparatus as set forth in claim 1,
wherein:

the turning state determining unit determines said
turning state from at least either one of a lateral
20 acceleration, a yaw rate or a steering angle.

6. The control apparatus as set forth in Claim 2,
wherein:

the turning state determining unit determines said
25 turning condition from at least either one of said lateral

acceleration, said yaw rate or said steering angle.

7. The control apparatus as set forth in Claim 3,
wherein:

5 the turning state determining unit determines said
turning condition from at least either one of said lateral
acceleration, said yaw rate or said steering angle.

8. The control apparatus as set forth in Claim 4,
10 wherein:

the turning state determining unit determines said
turning condition from at least either one of said lateral
acceleration, said yaw rate or said steering angle.

15 9. The control apparatus as set forth in Claim 5,
wherein:

the turning state determining unit determines that
the vehicle is in substantially a straight running state
when a first absolute value of at least either one of
20 the lateral acceleration, the yaw rate or the steering
angle is smaller than a predetermined value.

10. The control apparatus as set forth in Claim
6, wherein:

25 the turning state determining unit determines that

the vehicle is in substantially a straight running state when a first absolute value of at least either one of the lateral acceleration, the yaw rate or the steering angle is smaller than a predetermined value.

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11. The control apparatus as set forth in Claim 7, wherein:

the turning state determining unit determines that
10 the vehicle is in substantially a straight running state when a first absolute value of at least either one of the lateral acceleration, the yaw rate or the steering angle is smaller than a predetermined value.

15 12. The control apparatus as set forth in Claim 8, wherein:

the turning state determining unit determines that
the vehicle is in substantially a straight running state
when a first absolute value of at least either one of
20 the lateral acceleration, the yaw rate or the steering angle is smaller than a predetermined value.

13. A control method for four wheel drive vehicle having a differential limiting unit on at least either
25 one of a front wheel side or a rear wheel side, comprising

steps of:

determining the turning state of said vehicle;

calculating an actual left and right wheel
differential speed in accordance with a turning state
5 of the vehicle;

setting a left and right wheel target differential
speed;

setting a differential limiting torque at 0 on basis
of information from the differential limiting unit, when
10 an inside wheel speed falls below an outside wheel speed
by a preset threshold value in a turning state;

calculating a differential limiting torque on basis
of said information about the left and right wheel target
differential speed and the actual left and right wheel
15 differential speed, when said inside wheel speed exceeds
said outside wheel speed by a preset threshold value in
said turning state; and

setting a front and rear wheel differential limiting
torque calculated at the previous step in dependency on
20 said differential limiting torque on basis of said
information.

14. The control method as set forth in Claim 13,
wherein:

25 0 is the threshold value used for comparing the

inside wheel speed with the outside wheel speed in the turning state.

15. The control method as set forth in Claim 13,
5 wherein:

the actual left and right wheel differential speed calculating step regards an actual left and right wheel differential speed as an absolute value of said left and right wheel differential speed after the differential
10 limiting torque calculating step calculates a differential limiting torque based on said information from the differential limiting unit in dependency on the left and right wheel target differential speed and the absolute value when the turning state is determined said
15 turning state substantially as a running straight condition.

16. The control method as set forth in Claim 14,
wherein:

20 the actual left and right wheel differential speed calculating step regards an actual left and right wheel differential speed as an absolute value of said left and right wheel differential speed after the differential limiting torque calculating step calculates a
25 differential limiting torque based on said information

from the differential limiting unit in dependency on the
left and right wheel target differential speed and the
absolute value when the turning state is determined said
turning state substantially as a running straight
5 condition.

17. The control method as set forth in Claim 13,
wherein:

the turning state is determined said turning state
10 from at least either one of a lateral acceleration, a
yaw rate or a steering angle.

18. The control method as set forth in Claim 14,
wherein :

15 the turning state is determined said turning state
from at least either one of a lateral acceleration, a
yaw rate or a steering angle.

19. The control method as set forth in Claim 15,
20 wherein :

the turning state is determined said turning state
from at least either one of a lateral acceleration, a
yaw rate or a steering angle.

25 20. The control method as set forth in Claim 16,

wherein :

the turning state is determined said turning state from at least either one of a lateral acceleration, a yaw rate or a steering angle.

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21. The control method as set forth in Claim 17, wherein:

the turning state is determined that the vehicle is in substantially a straight running state when a first
10 absolute value of at least one of the lateral acceleration, the yaw rate or the steering angle is smaller than a predetermined value.

22. The control method as set forth in Claim 18,
15 wherein:

the turning state is determined that the vehicle is in substantially a straight running state when a first absolute value of at least one of the lateral acceleration, the yaw rate or the steering angle is smaller than a
20 predetermined value.

23. The control method as set forth in Claim 19, wherein:

the turning state is determined that the vehicle
25 is in substantially a straight running state when a first

absolute value of at least one of the lateral acceleration,
the yaw rate or the steering angle is smaller than a
predetermined value.

5 24. The control method as set forth in Claim 20,
wherein:

the turning state is determined that the vehicle
is in substantially a straight running state when a first
absolute value of at least one of the lateral acceleration,
10 the yaw rate or the steering angle is smaller than a
predetermined value.